

PERFORMANCE AND EMISSION CHARACTERISTICS OF DI ENGINE BY USING CRUDE OIL

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ABSTRACT

In this study, a non-consumable vegetable oil was delivered from Crude organic products as a substitute fuel for diesel motors and its ease of use was examined as immaculate oil and as a mix with diesel fuel. An immediate infusion (DI) diesel motor was tried utilizing diesel, Crude oil, and mixes of this oil and diesel in various extents. An extensive variety of motor burdens and Crude oil/diesel proportions of 5/95% (J5), 10/90% (J10), 20/80% (J20), 50/half (J50), and 80/20% (J80) by volume were considered. The accompanying execution parameters were measured; brake warm effectiveness, brake particular fuel utilization and CO and CO₂ emanations. No critical change in brake warm effectiveness and brake particular fuel utilization was experienced up to J20 proportions. Be that as it may, higher mixes experienced disintegration in productivity and fuel utilization around 10 to 25%. At less load operations, CO₂ outflow with mixes was lower than diesel, though, at high loads, CO₂ emanation got to be higher with a greater rate of this oil in the mixes. In any case, CO emanation with mixes was much greater than that of diesel; the higher the rate of Crude oil in the mix, the higher the CO discharge.

Keyword: Crude oil, performance and emission, characteristics, various blends, computerised 5 gas analyser, diesel engine.

1.INTRODUCTION

Vitality interest is expanding because of the expanding number of vehicles. Fossil fills are restricted assets; henceforth, scan for renewable powers are turning out to be increasingly unmistakable for guaranteeing vitality security and natural insurance. As per International Energy viewpoint, 2007 distributed by the Energy Information Administration, the world utilization for petroleum and other fluid fuel will develop from 83 million barrels/day in 2004 to 97 million barrels/day in 2015 and a little more than 118 million barrels/day in 2025. Under these development suspicions, roughly half of the world's aggregate assets would be depleted by 2025. Also, numerous studies evaluating that the world oil creation would crest at some point between 2007 and 2025. Therefore the future vitality accessibility is a major issue for us. Fills got from reusable organic assets for use in diesel engines are known as biodiesel. Biodiesel is a fuel containing 10% to 15% oxygen. The oil substance of Crude seed reaches to 55% by its weight. The unsaturated fat structure of Crude it as a linoleic or oleic corrosive sort, which are unsaturated fats. The unsaturated fat creation of Crude oil comprises of myristic, palmitic, stearic, arachidic, oleic and linoleic acids. The seed and oil are dangerous because of the closeness of curative and curative. However, from the properties of this oil, it is conceived that the oil would be appropriate as fuel oil.

2.LITERATURE REVIEW

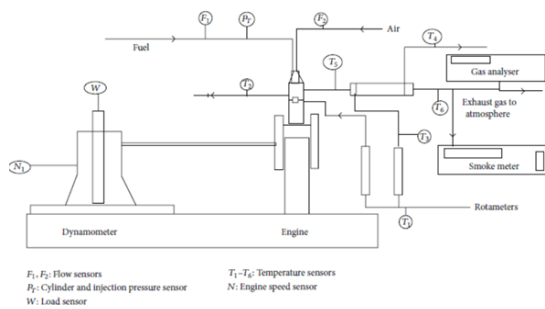
Vijitra Chalatlal¹, Murari Mohon Roy²: The higher warming estimation of Crude oil is 85% to that of diesel and pour point is fundamentally the same as. The glimmer point and kinematic consistency are a few higher than that of diesel. Steady speed motor test were completed utilizing this oil. The execution of the engine was assessed as far as brake warm productivity, brake fuel utilization and fumes gas temperature. The brake force is expanded from 0kw for no heap up to around 16.75kw for the full load with various fills. The most extreme warm productivity is 30%. J5 showed marginally higher warm effectiveness than diesel. J10 and J20 indicated comparative warm proficiency however J50 and higher mixes demonstrated 3-5% less warm effectiveness than diesel. The BFSC diminished from 0.87kg/kwh at less load to 0.28 kg/kwh at high load for diesel. The best BSFC acquired for diesel is 0.28 kg/kwh at full load operation better BSFC is gotten from J5. The abundance air variable is diminished from 5.75 at no heap to 1.75 at high load for diesel. The most minimal estimation of RMSE brake fuel utilization is 0.00012 for J5 and J10. The RMSE amongst diesel and diverse Crude oil mixes that J5 to J20 have the least estimation of RMSE of brake warm productivity (0.0001). This implies that this oil has 10% motor execution.

N. Manikanda Prabu, Dr. S. Nallusamy: At higher yield conditions because of a higher crest and fumes temperatures the NOX qualities are moderately higher contrasted with lower power yield conditions.

The CO₂ expanded regarding expanded of burden conditions for both the diesel and Crude oil. The expansion of Crude oil is reasonable and eco inviting one this is thought to be upgraded mix.

Prashanth Gill, S.k Soni, K.Kundu: The specific fuel utilization (SFC) of the HOME & JOME and in addition diesel mixes diminished with expanding load. The CO₂ outflow from the diesel motor with various mixes was about expanding of CO₂ concerning the expanding of burden conditions. As burden increments, there was a continuous diminishing in CO emanation from the bio-fuel as for diesel. At higher force yield conditions because of higher fumes temperatures the NO_x qualities are moderately higher contrasted with lower power yield conditions. Smoke increments with expansions of burden. The motor works easily on methyl ester of hemp oil and Crude oil with execution practically identical to diesel operations.

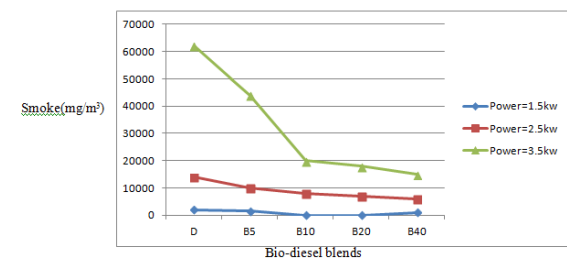
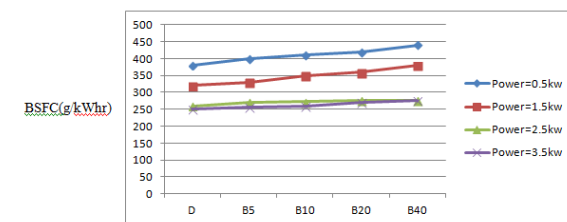
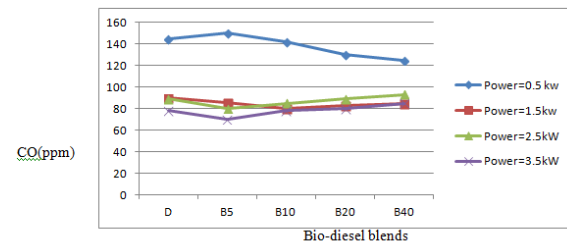
3. EXPERIMENTAL SETUP

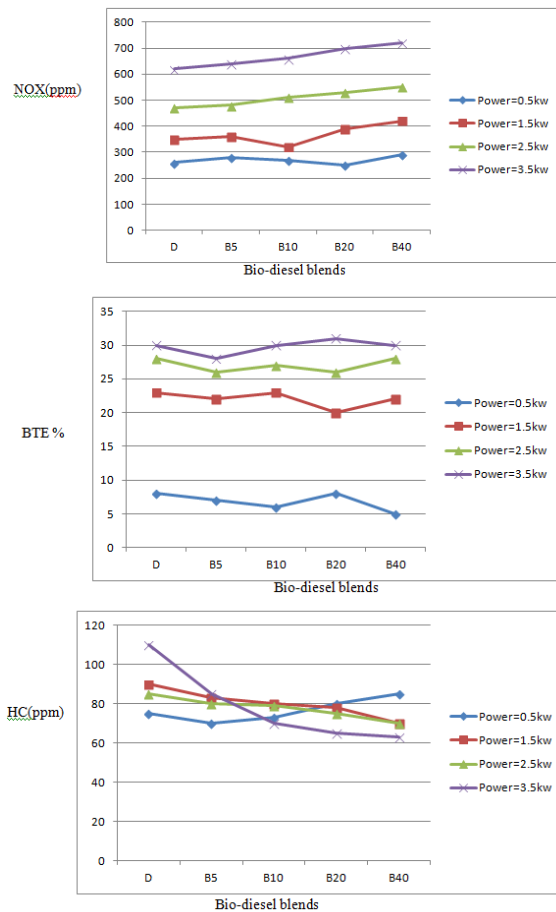


In the present research work 3.5kW single-cylinder 4-stroke water-cooled diesel engine at 1500 rpm was used. The schematic of the experimental set up is shown in the above Figure. We have used an eddy current dynamometer for load control on the engine. The piezoelectric transducer was mounted on cylinder head. Several thermocouple temperature sensors were installed at specific locations to measure inlet and outlet water, manifold air temperature, exhaust outlet, and heat exchanger outlet temperatures. A thermocouple was fixed on the surface of high pressure fuel pipe to measure the temperature. A crank angle encoder was linked with the engine shaft. Dual openings were made in exhaust gas pipeline for sampling. A burette fitted with a 3 way valve meters the fuel. The rate of intake air was measured with an orifice meter connected to a manometer. A waste tank was used to overcome the pulsations generated by the engine, for ensuring a steady air intake through the intake manifold. An AVL 444 Di gas analyser measures the CO, HC, and NO_x emissions. The AVL 437 measures the density of the exhaust smoke. The engine was warmed up earlier to data acquisition. All the engine test runs

were carried out under constant ambient conditions. During the tests, first the engine was started with diesel to warm up and then fuel was switched. After finishing the tests with diesel-biodiesel blends, the engine was shifted to diesel and the engine was operated until the biodiesel had been drained from the fuel line, injection pump, and the injector. This was carried out to overcome the starting difficulties at the later time. Initially the engine was started with diesel for 10 minutes to warm up. After this the initial data was generated and the corresponding results were obtained. Every time the engine speed was checked and maintained constant. All the measurements were taken for five times, and the mean of these readings was used for calculation and analysis. The various parameters analyzed in the present investigation were brake thermal efficiency (BTE), brake specific fuel consumption (BSFC), exhaust gas temperature (EGT), carbon monoxide (CO), unburned hydrocarbons (UHC), nitrogen oxides (NO_x), and smoke opacity. For base data, the engine compression ratio was set to 17.5 and fuel injection pressure (P_{inj}) was maintained at 180 bar. Then compression ratio and fuel injection pressure were increased from 17.5 to 19.5 in the steps of 1 and from 180 bar to 220 bar in the step of 20 bar respectively.

4. RESULTS AND DISCUSSION





The Emissions of various Biodiesel Blends:

Figure shows the variation of exhaust smoke opacity for all biodiesel blends at various loads. The opacity was reduced from 19.4% to 44.4% for various blends as compared to diesel. This was mainly due to complete combustion, as more amount of oxygen is available from fuel itself. The smoke opacity increases as engine loads increases. It was observed that the smoke increased even at lower loads and was higher than mineral diesel. This is due to very bad atomization of Crude oil. High molecule weights, higher viscosity, and low volatility result in poor atomization of fuel. Figure 3 shows the variation of NO_x emission at various load points and for various blends. NO_x was increased from 2.6% to 16.2% for various biodiesel blends at different load conditions as compared to diesel. As the engine load is increased, the mass emission of NO_x reduces. The most important reason for the formation of NO_x is the combustion temperature in the cylinder and the availability of oxygen. The Crude oil has considerable higher viscosity which results into longer combustion duration and demonstrates significant energy release during the last phase of burning. Figs 4 & 5 show the variation of CO and THC emission trends at various loads. Continuous reduction in CO was observed at 0.5 kW, 1.5 kW, 2.5 kW load whereas at 3.5kW load and B20 blends CO emission increases. An increase in the HC emissions was noticed at full load condition with

increased biodiesel blends. CO emissions were reduced to 9.33% at full load. The HC emission was reduced to 4.1% and 6.66% for B5 and B10 blends and then increases for B20 to B100 blends. With an increase in Biodiesel blends, the energy content in the fuel reduces. As a result to get the same power, more quantity of fuel needs to be injected and there are chances of wall wetting and fuel trapped in specific zones may increase the HC emissions. Blends higher than 20% showed higher CO emissions.

REFERENCES:

[1].Bari S, Roy MM (1995). "Prospect of rice bran oil as an alternative to diesel", Proc. of the Fifth Int. Conf. on small engines, their fuels and the environment. The University of Reading, UK, pp. 31-36.
 [2].Çetin M, Yüksel F (2007). The use of hazelnut oil as a fuel in prechamber diesel engine. Appl. Therm. Eng. 27: 63-67.
 [3].Chauhan BS, Kumar N, Jun YD, Lee KB (2010). Performance and emission study of preheated Crude oil on medium capacity diesel engine. Energy, 35: 2484-2492.
 [4].Agarwal AK (1998). Vegetable oils versus diesel fuel: development and use of biodiesel in a compression ignition engine. TIDE, 8(3):191-204.
 [5].Hazar H, Aydin H (2010). Performance and emission evaluation of a CI engine fueled with preheated raw rapeseed oil (RRO)-diesel blends. Appl. Energy, 87(3): 786-790.
 [6].He Y, Bao YD (2005). Study on cottonseed oil as a partial substitute for diesel oil in fuel for single-cylinder diesel engine. Renewable Energy, 30: 805-813.
 [7].Kalam MA, Husnawan M, Masjuki MH (2003). Exhaust emission and combustion evaluation of coconut oil-powered indirect injection diesel engine. Renewable Energy, 28: 2405-2415.
 [8].Kpikpi WM (2002). Crude as vegetable source of renewable energy. Paper Presented at ANSTI Sub-network Meeting on Renewable Energy, pp. 18-22.
 [9].Kumar MS, Ramesh A, Nagalingam B (2003). An experimental comparison of methods to use methanol and Crude oil in a compression ignition engine, Biomass Bioenerg., 25: 309-318. No SY (2011). Inedible vegetable oils and their derivatives for alternative diesel fuels in CI engines: A review. Renewable Sustain. Energy Rev., 15: 131-149.